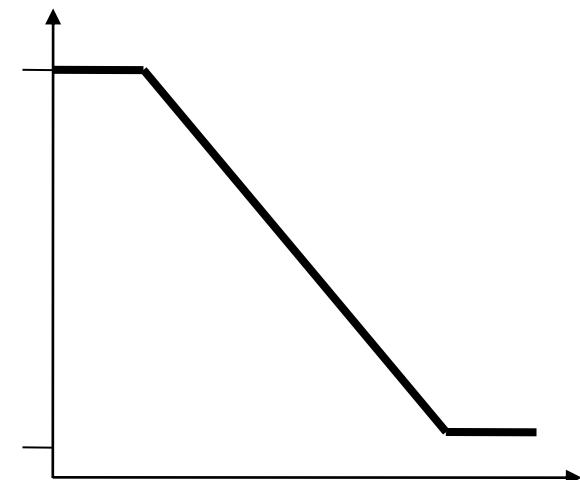
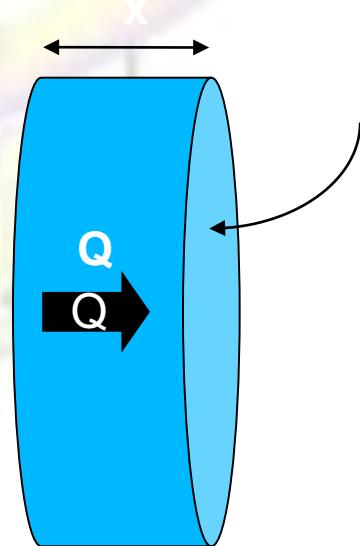


# Heat Transfer

- Thermal conduction
- Heat Exchanger Types
  - Parallel / counter/ cross flow
  - Double pipe
  - Shell and tube
  - Plate / Compact
  - Single / multi pass
- Heat Exchanger Analysis
  - Conservation of heat
  - Log mean temperature difference
  - Effectiveness / NTU
  - Fouling
  - Heat Exchanger Design

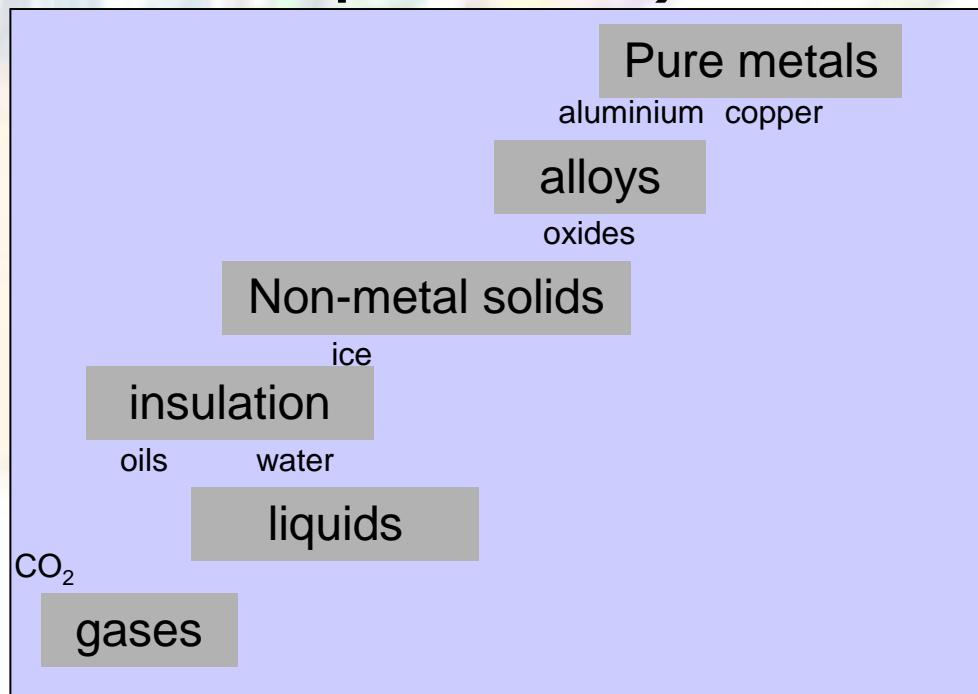
# Thermal Conduction

- In all our heat exchangers we have two fluids at different temperatures separated by a wall
- Heat transferred by conduction across the wall
- Remember from 1<sup>st</sup> year Physics;  $Q/t = kA(T_2 - T_1)/x$
- Differential form:
  - $Q/t = kA dT/dx$



# Thermal Conductivity

- Property of material
- Units:  $\text{Wm}^{-1}\text{K}^{-1}$
- Depends on temperature (decreases for most metals with temperature)



0.01 0.1 1 10 100 1000  
Thermal conductivity,  $k$ , ( $\text{Wm}^{-1}\text{K}^{-1}$ )

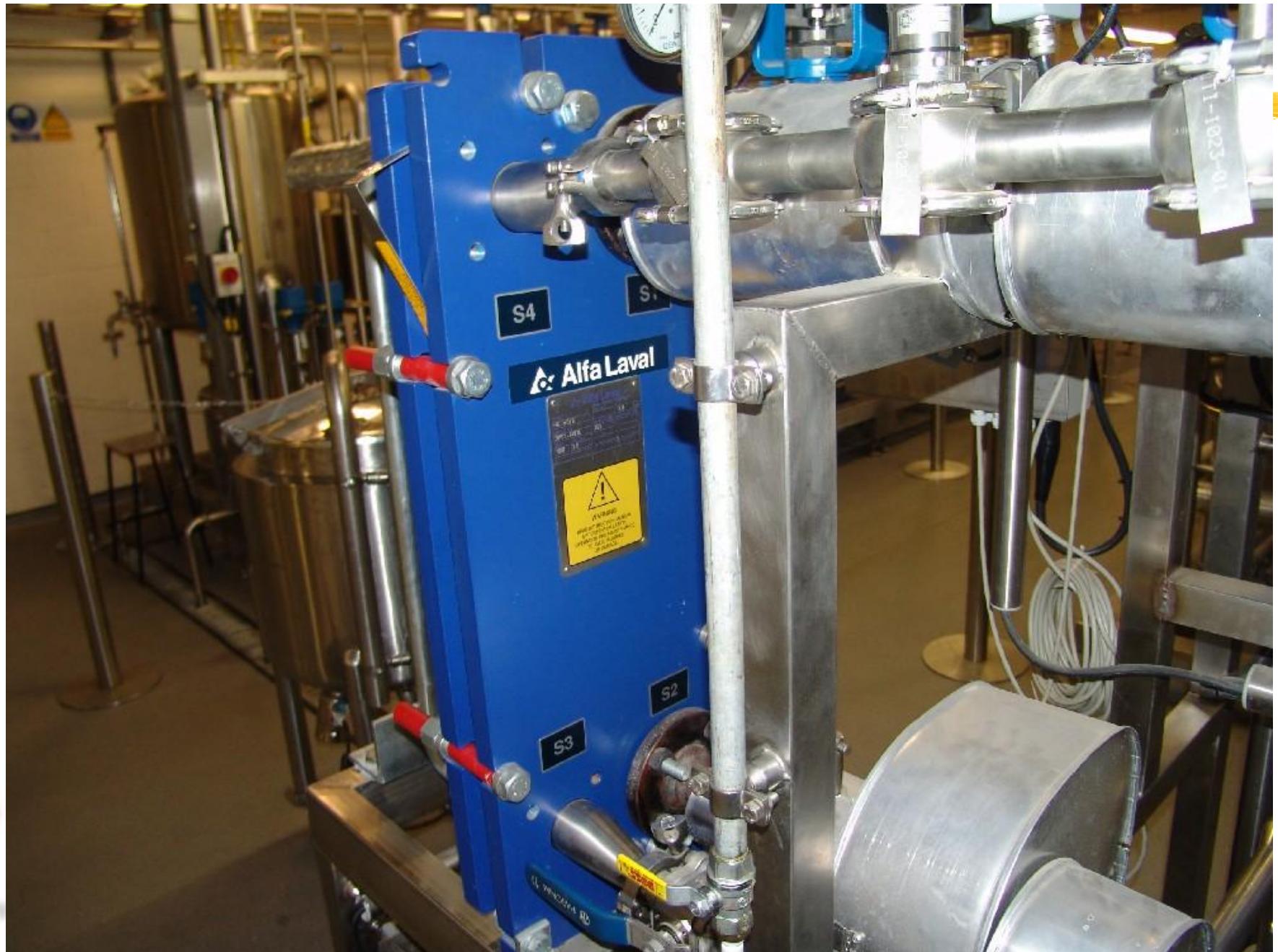
# Example

- An aircraft window is  $0.1\text{m}^2$ . It is 10mm thick and made from polycarbonate ( $k_{\text{pc}} = 0.21\text{Wm}^{-1}\text{K}^{-1}$ ). The temperature difference across the window is  $80^\circ\text{C}$ . Calculate the heat loss through the window, the total heat loss if there are 200 windows on the plane, and the heating bill on a four hour flight if it costs  $\text{€1kWhr}$ .

**[168W, 33.6kW, €134.4]**

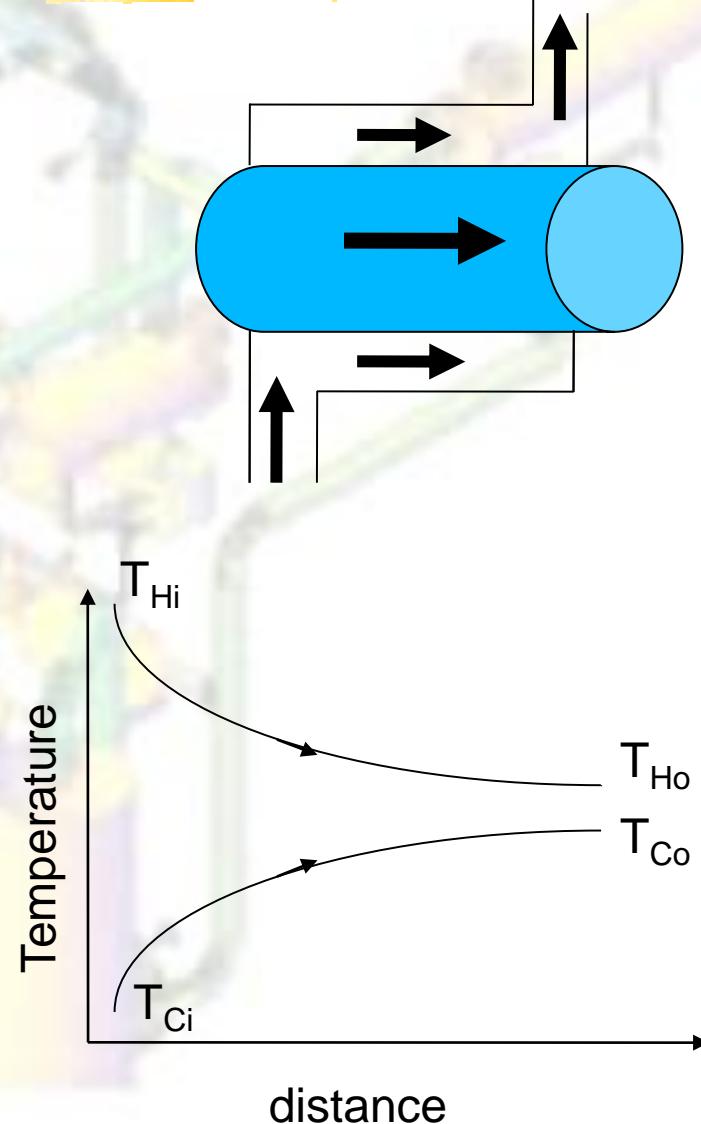
# Heat Exchangers

- Two fluids at different temperatures in thermal contact
- Separated by solid wall
- Heat transferred by conduction through the wall and then convection through the fluid
- The convection works best if flow turbulent
  - Specify minimum flow rate (say 3m/s)
  - Often used in regeneration mode; hot fluid leaving a reactor vessel heats incoming fluid



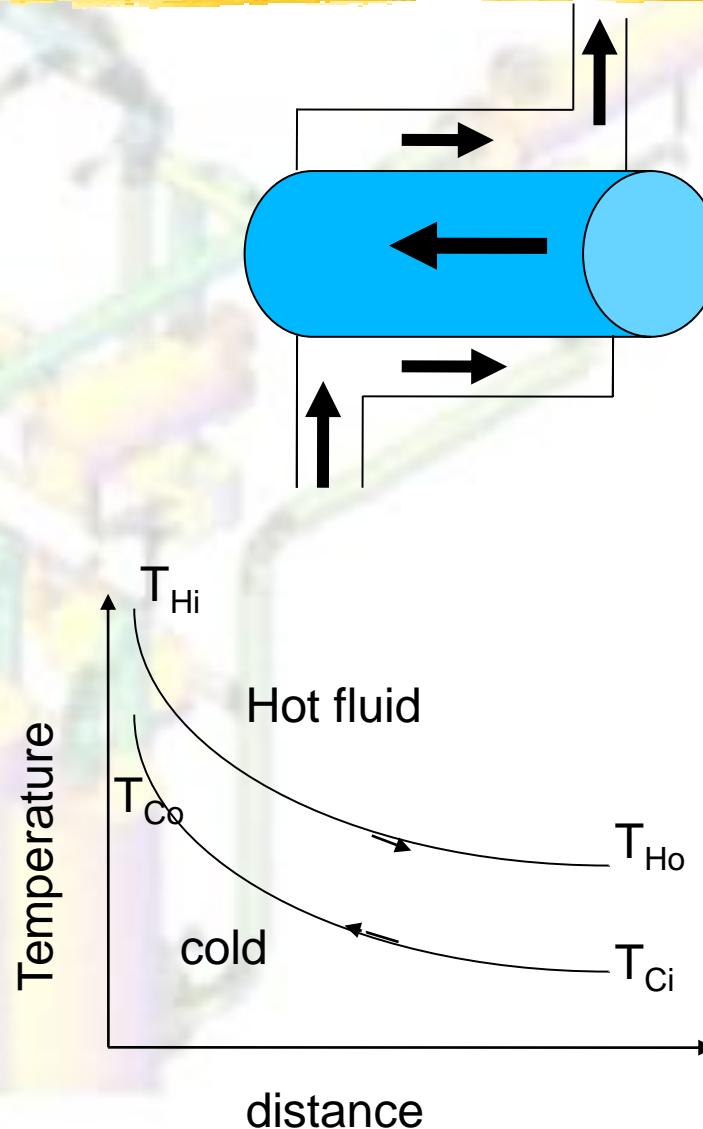
# Parallel / Counter / Cross Flow

- Parallel Flow
  - two fluids flow in same direction



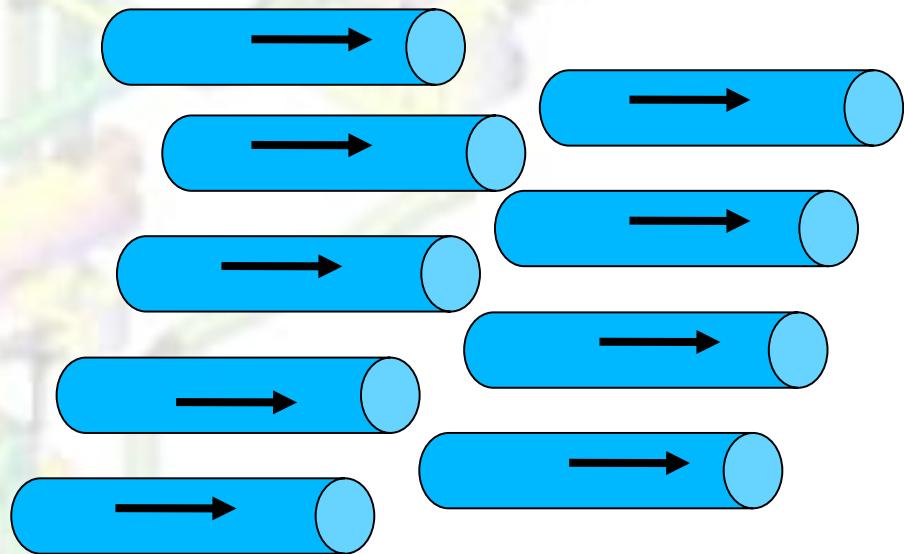
# Parallel / Counter / Cross Flow

- Counter Flow
  - two fluids flow in opposite directions

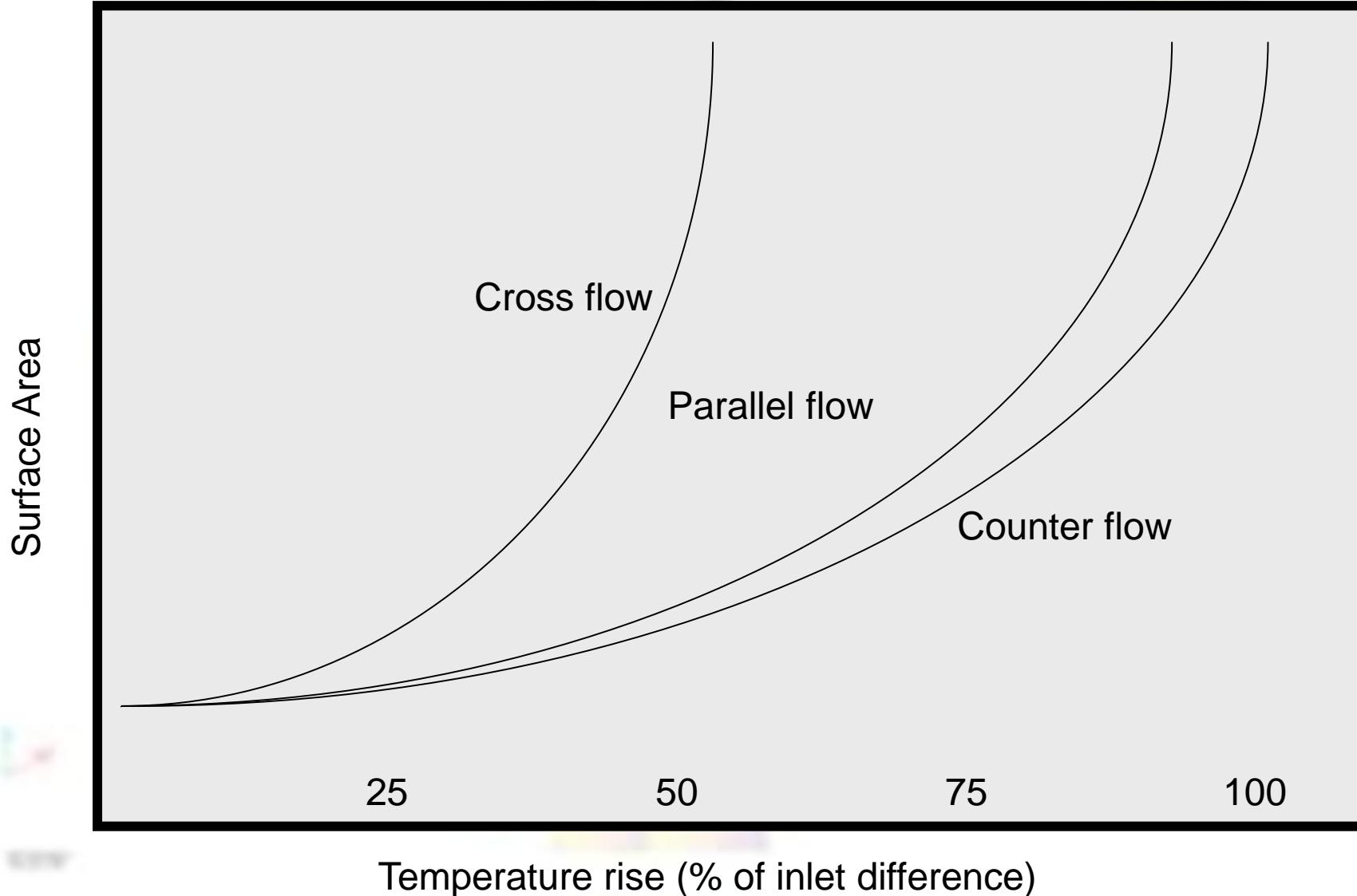


# Parallel / Counter / Cross Flow

- Cross Flow – two fluids flow at right angles to each other



# How Different Flow Patterns Compare



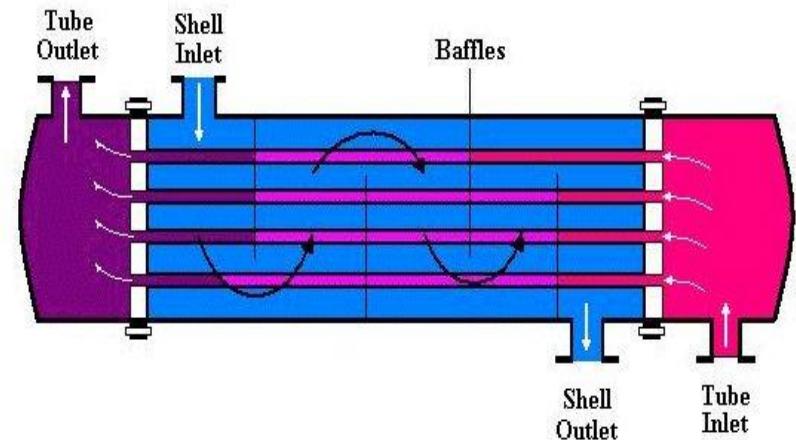
# Double Pipe Heat Exchanger

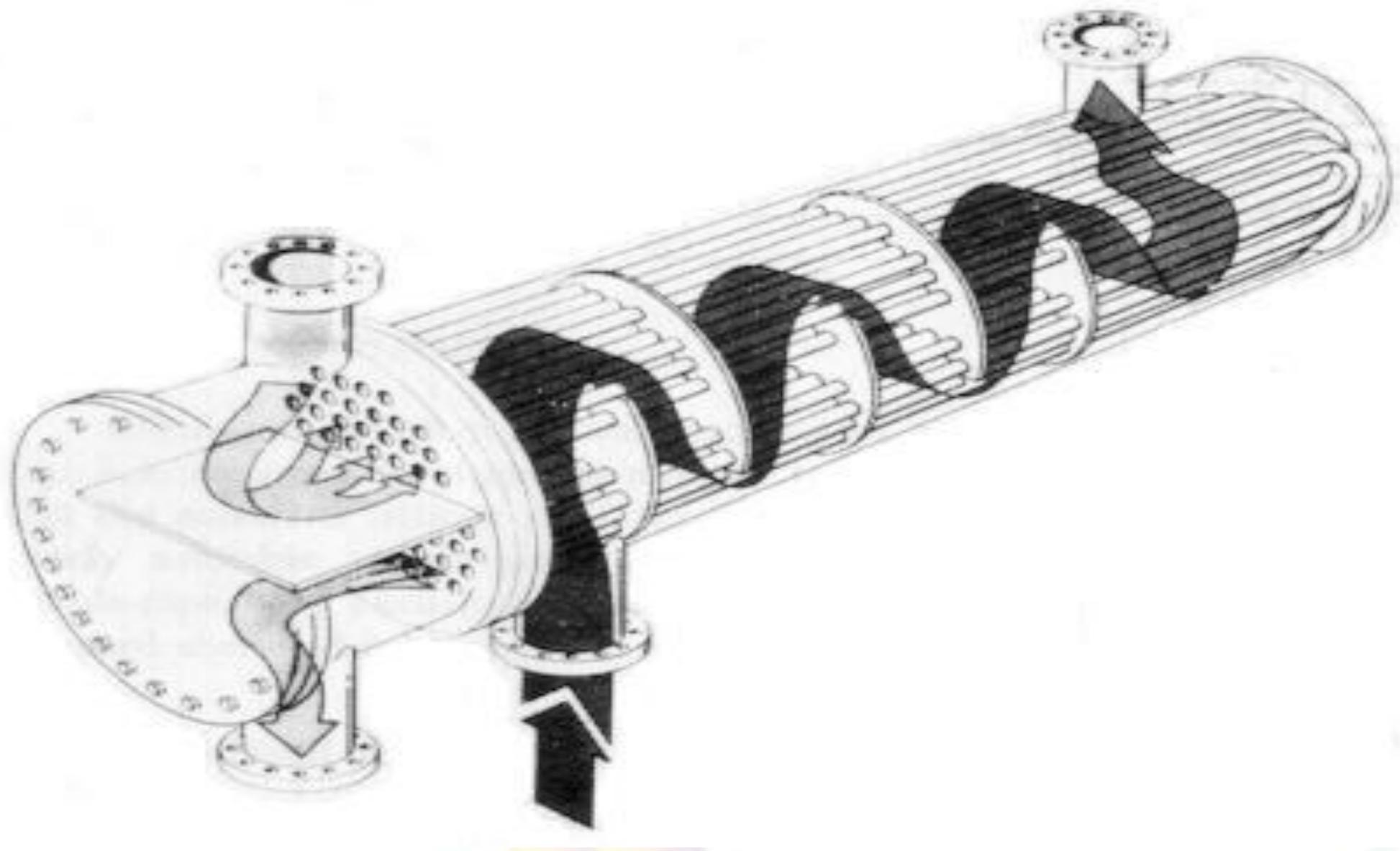
- One pipe located concentrically inside a second, larger one
- Have to be large to be effective



# Shell & Tube Exchangers

- Most common HE (80% of market)
- One fluid in network of tubes, second surrounding in shell
- Baffles to complicate flow, improve efficiency (straight, helical)
- If phase change, should be in shell
- Tubes usually doubled or quadrupled back (input and output at same end)
- Tubes ribbed on inner and outer surfaces

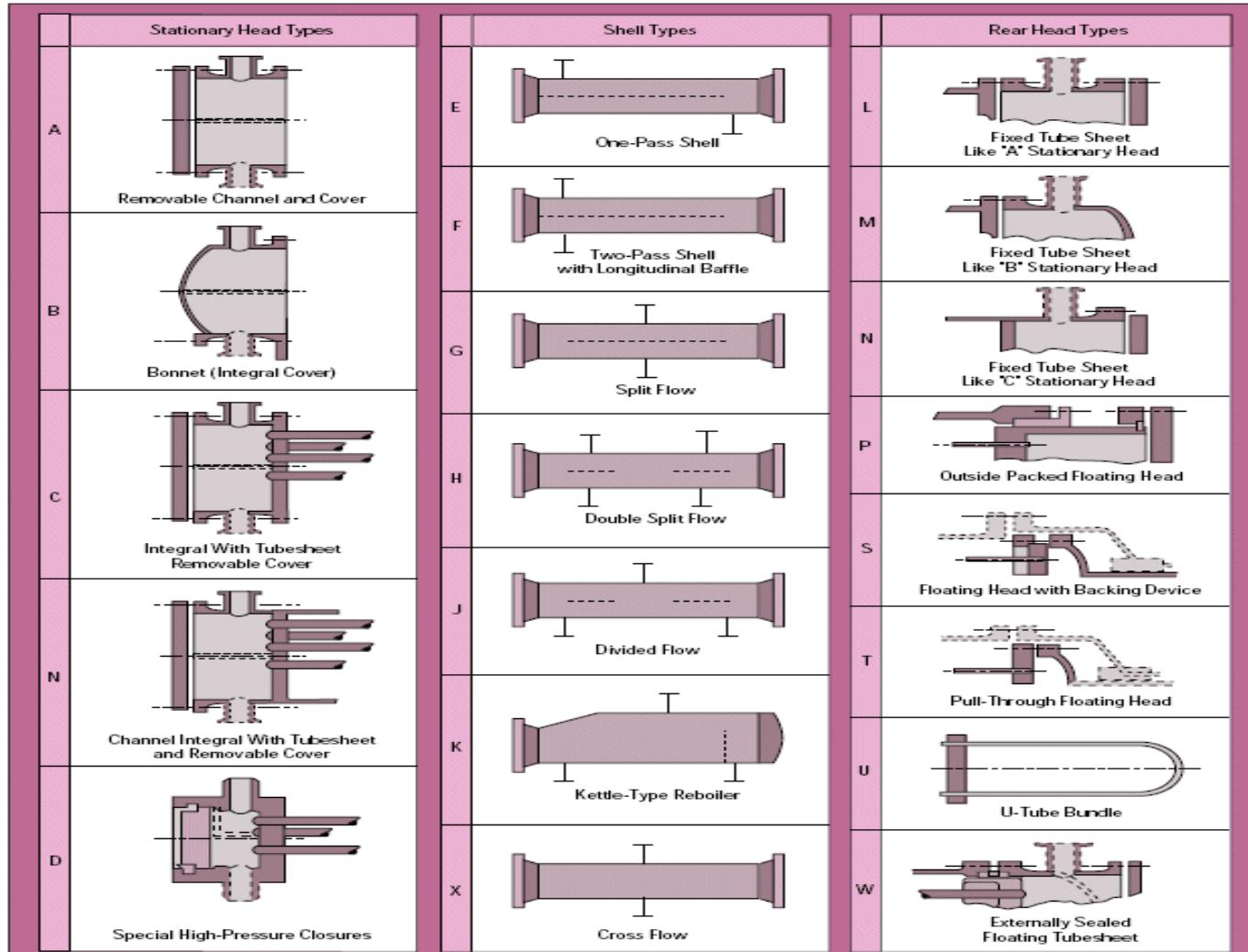




# Shell & Tube Exchangers

- Specified by three letter code from TEMA
- Put high pressure (up to 100MPa) or corrosive fluids in tubes
- Put fluids prone to fouling in tubes (higher velocity = less build-up)
- Get lower pressure drop in tubes
- Vibration can be problem – avoid resonance
- Temperature stresses important – U shaped bundles good

# 3 Letter TEMA Codes



Source:

CHEMICAL ENGINEERING PROGRESS • FEBRUARY 1998

# Heat Exchanger Analysis

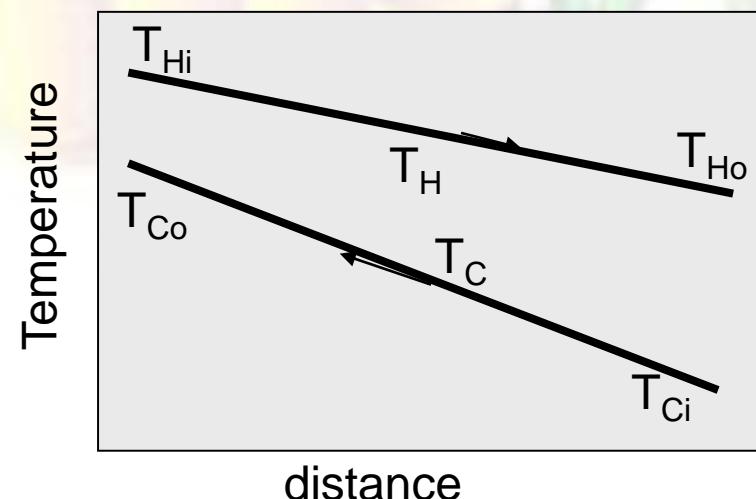
- Four temperatures
- Two flow rates
- Two heat capacities
- Conductivity of separating wall
- Area of separating wall
- Fouling
- Type of flow (cross/parallel/counter)

# Conservation of Heat

- Heat lost by hot fluid = heat gained by cold fluid
- Heat change
  - $Q = mc\Delta T$  for change in temperature
  - $Q = mL_{v,f}$  for change of phase
- $mc = C$

# Log Mean Temperature Difference

- For conduction –  $Q/t = UA \Delta T$  where  $\Delta T$  is the difference in temperatures
- But for a heat exchanger  $\Delta T$  changes along the length of the device
- For example, for a counter flow the profile is shown below
- So what value of  $\Delta T$  should we use in the equation?



$$\Delta T_{lm} = \frac{[(T_{H_o} - T_{C_i}) - (T_{H_i} - T_{C_o})]}{\log_e \frac{(T_{H_o} - T_{C_i})}{(T_{H_i} - T_{C_o})}}$$

(counter flow)

- is known as the log mean temperature difference.
- It reflects an average value of the temperature difference across the Heat Exchanger
- For crossflow or multipass heat exchangers we use the same expression but introduce a correction term:

$$\Delta T_{lm} = F \frac{[(T_{H_o} - T_{C_i}) - (T_{H_i} - T_{C_o})]}{\log_e \frac{(T_{H_o} - T_{C_i})}{(T_{H_i} - T_{C_o})}}$$

(cross flow)

- F usually found from graphs (use F = 0.9 when graphs not available)
- For parallel flow the equation is:

$$\Delta T_{lm} = \frac{[(T_{H_o} - T_{C_o}) - (T_{H_i} - T_{C_i})]}{\log_e \frac{(T_{H_o} - T_{C_o})}{(T_{H_i} - T_{C_i})}}$$

(parallel flow)

- In general get:

$$\Delta T_{lm} = F \frac{\Delta T_2 - \Delta T_1}{\log_e \frac{\Delta T_2}{\Delta T_1}}$$

- Where  $\Delta T_2$  is the temperature difference at one end of the pipe,  $\Delta T_1$  is the temperature difference at the other end and  $F$  is a correction factor read from tables

# Problem on LMTD

- A parallel flow double pipe heat exchanger heats 2.52kg/s of water from 21.1°C to 54.4°C using hot water under pressure entering at 110°C and leaving at 70°C. The heat exchanger area is  $A = 4.3\text{m}^2$  and the heat capacity of water is  $4180\text{Jkg}^{-1}\text{°C}^{-1}$ . Calculate the  $\Delta T_{lm}$  in the exchanger and the overall heat transfer coefficient  $U$ .

**[42.1°C, 1938Wm<sup>-2</sup>K<sup>-1</sup>]**

- A counter flow double pipe heat exchanger heats water from 15°C to 74.2°C using hot water entering at 91.3°C and leaving at 62.2°C. The heat exchanger area is  $A = 8.51\text{m}^2$  and the overall heat transfer coefficient  $U = 2500\text{Wm}^{-2}\text{K}^{-1}$ . Calculate the  $\Delta T_{lm}$  in the exchanger,  $q$ , and the flow rates in both pipes.

**[29.3°C, 6.25x10<sup>5</sup>W, 2.52kg/s for the cold water, 5.14kg/s for the hot]**

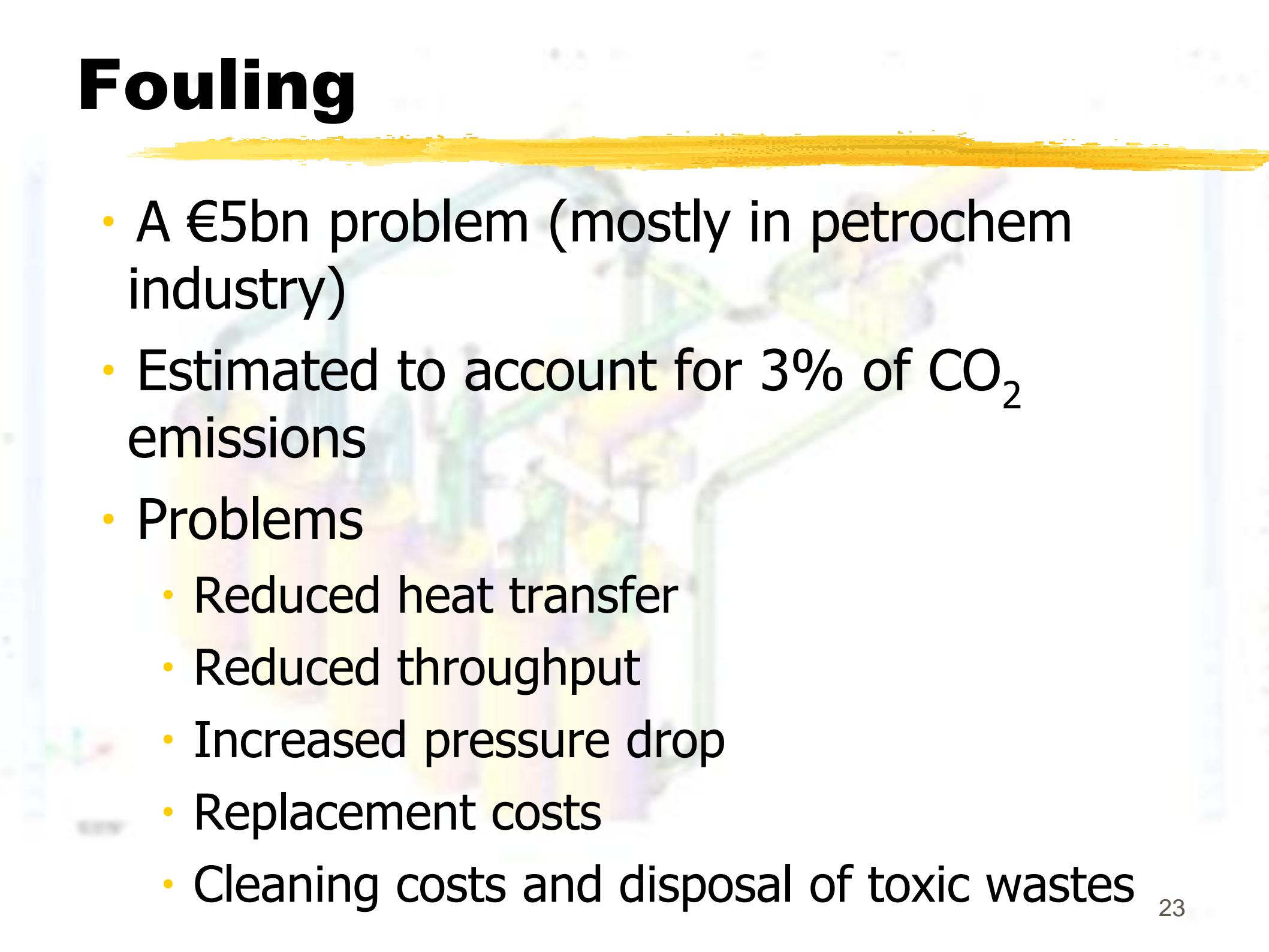
- A parallel flow double pipe heat exchanger heats water from 24°C to 44.9°C using hot water entering at 95°C. The heat exchanger area is  $A = 4.4\text{m}^2$ . The flow rates are 1.14kg/s for the cold water and 3.1kg/s for the hot water. Calculate the temperature of the hot water leaving the exchanger, the LMTD, and the overall heat transfer coefficient  $U$ .

**[83.8°C, 53.4°C, 4238Wm<sup>-2</sup>K<sup>-1</sup>]**

# Fouling



# Fouling



- A €5bn problem (mostly in petrochem industry)
- Estimated to account for 3% of CO<sub>2</sub> emissions
- Problems
  - Reduced heat transfer
  - Reduced throughput
  - Increased pressure drop
  - Replacement costs
  - Cleaning costs and disposal of toxic wastes

# Fouling - Causes

- Dirt, soot scale
- Coke from petroleum industry
- Corrosion products
- Algae and other biological growths (prevented by copper)
- Sedimentation – if fluid is a suspension.  
Self limiting
- Inverse solubility – as fluid cools down
- Chemical reactions

# Fouling - Prevention

- Avoid low fluid velocities (less than 1m/s)
- Avoid high fluid velocities (greater than 4m/s for water). Causes erosion
- Avoid big temperature differences