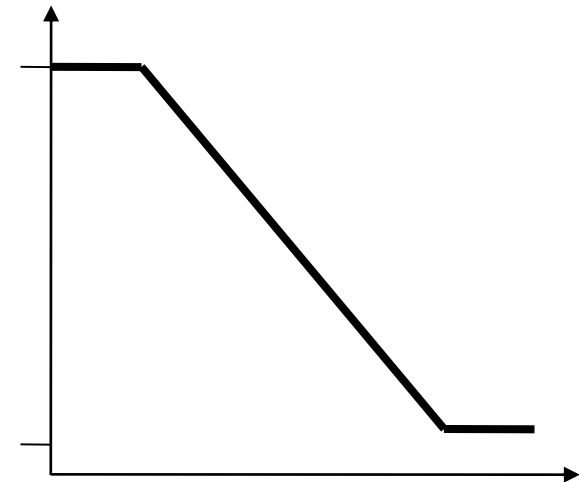
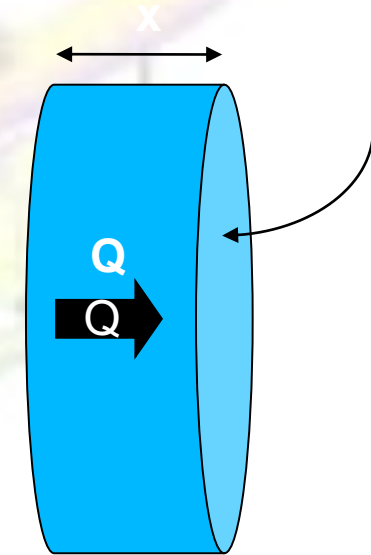


Heat Transfer

- Thermal conduction
- Heat Exchanger Types
 - Parallel / counter/ cross flow
 - Double pipe
 - Shell and tube
 - Plate / Compact
 - Single / multi pass
- Heat Exchanger Analysis
 - Conservation of heat
 - Log mean temperature difference
 - Effectiveness / NTU
 - Fouling
 - Heat Exchanger Design

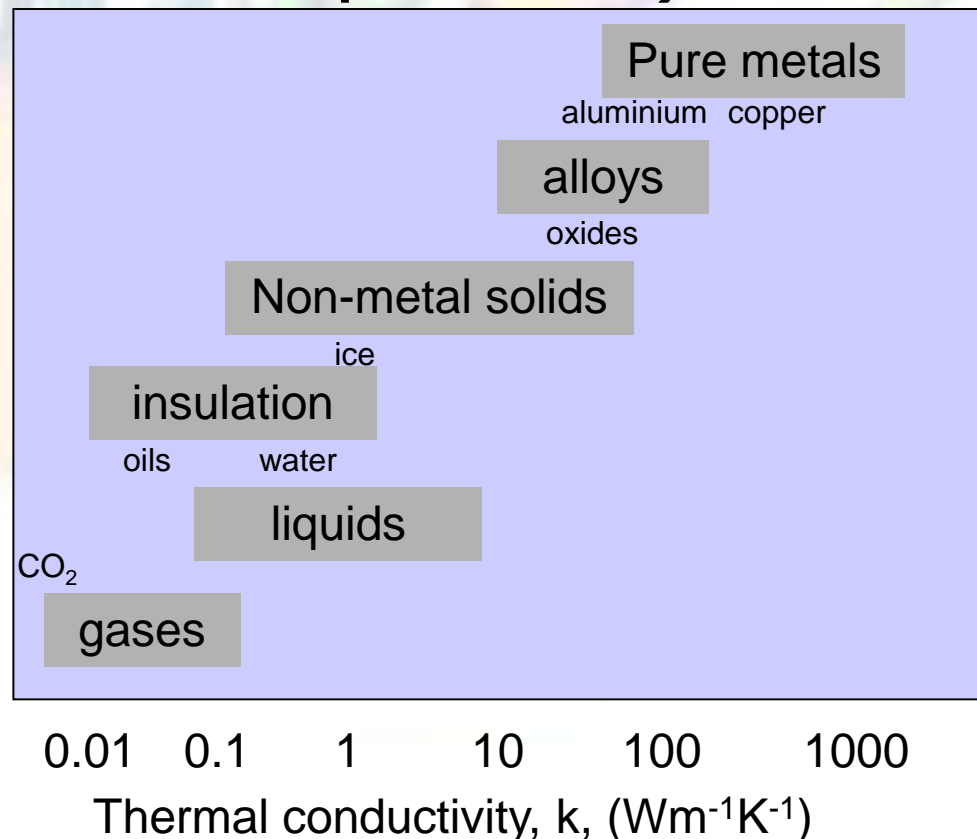
Thermal Conduction

- In all our heat exchangers we have two fluids at different temperatures separated by a wall
- Heat transferred by conduction across the wall
- Remember from 1st year Physics; $Q/t = kA(T_2 - T_1)/x$
- Differential form:
 - $Q/t = kA \, dT/dx$



Thermal Conductivity

- Property of material
- Units: $\text{Wm}^{-1}\text{K}^{-1}$
- Depends on temperature (decreases for most metals with temperature)



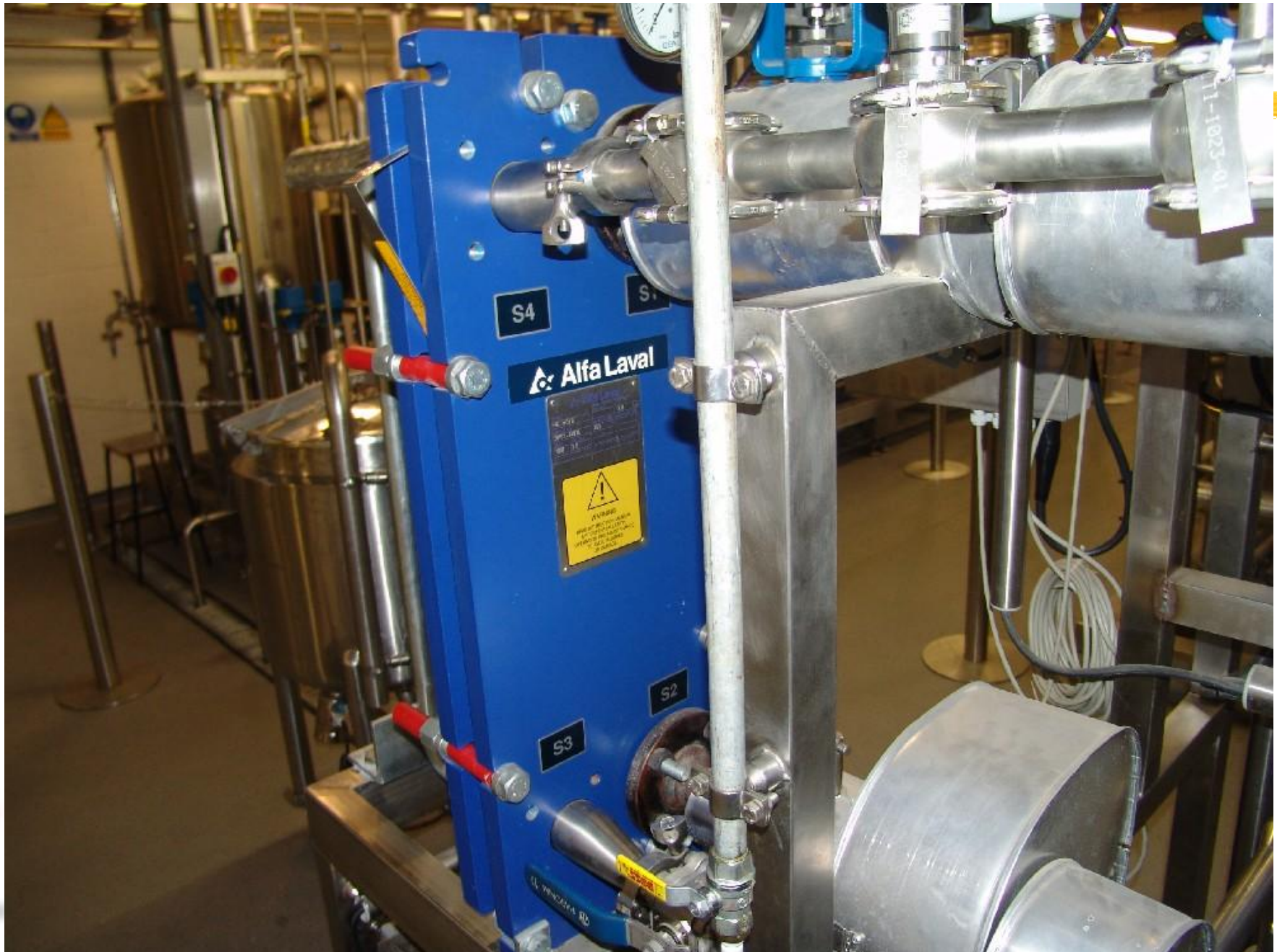
Example

- An aircraft window is 0.1m^2 . It is 10mm thick and made from polycarbonate ($k_{\text{pc}} = 0.21\text{Wm}^{-1}\text{K}^{-1}$). The temperature difference across the window is 80°C . Calculate the heat loss through the window, the total heat loss if there are 200 windows on the plane, and the heating bill on a four hour flight if it costs $\text{€}1\text{kWhr}$.

[168W, 33.6kW, €134.4]

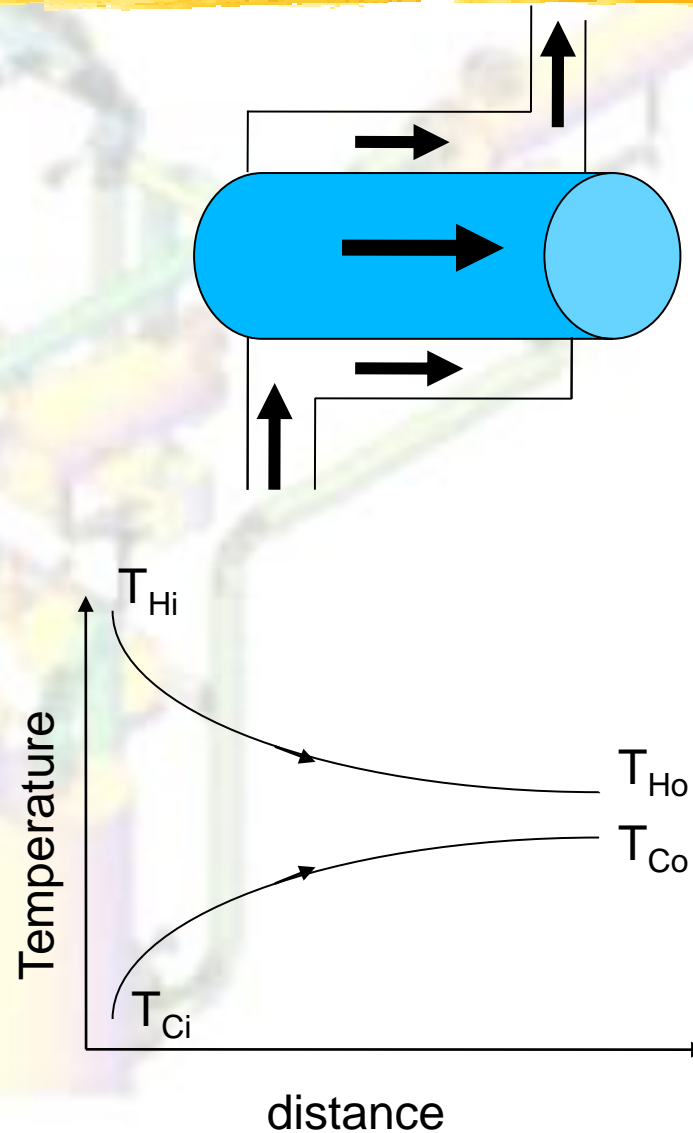
Heat Exchangers

- Two fluids at different temperatures in thermal contact
- Separated by solid wall
- Heat transferred by conduction through the wall and then convection through the fluid
- The convection works best if flow turbulent
 - Specify minimum flow rate (say 3m/s)
- Often used in regeneration mode; hot fluid leaving a reactor vessel heats incoming fluid



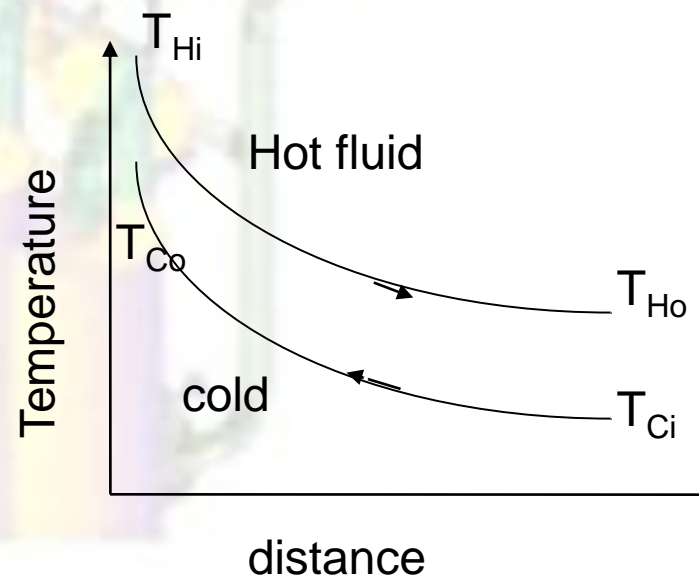
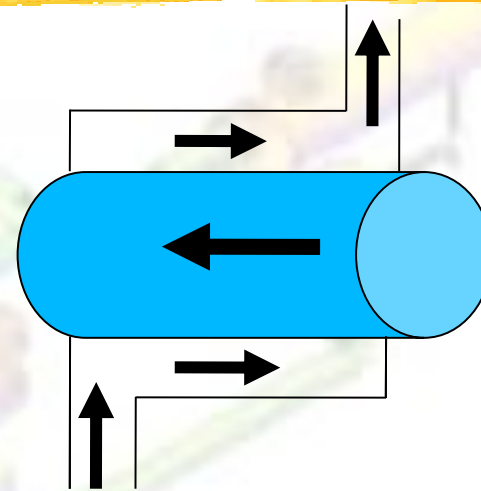
Parallel / Counter / Cross Flow

- Parallel Flow
– two fluids flow in same direction



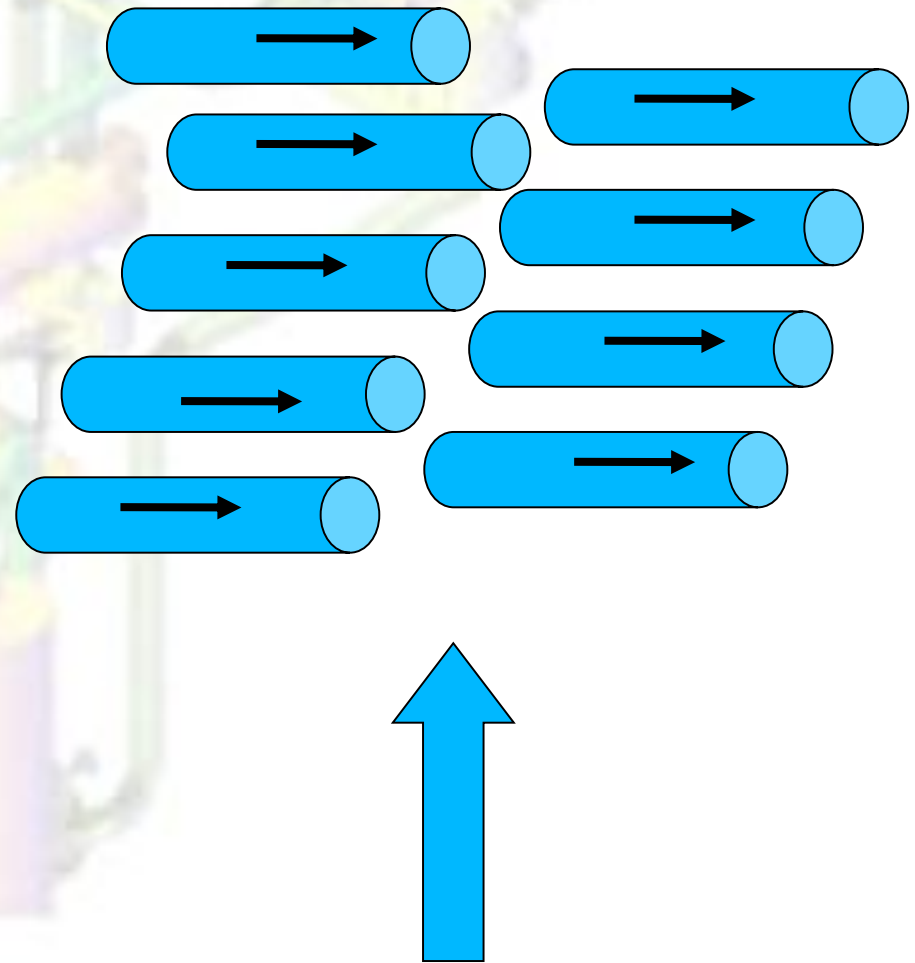
Parallel / Counter / Cross Flow

- Counter Flow – two fluids flow in opposite directions

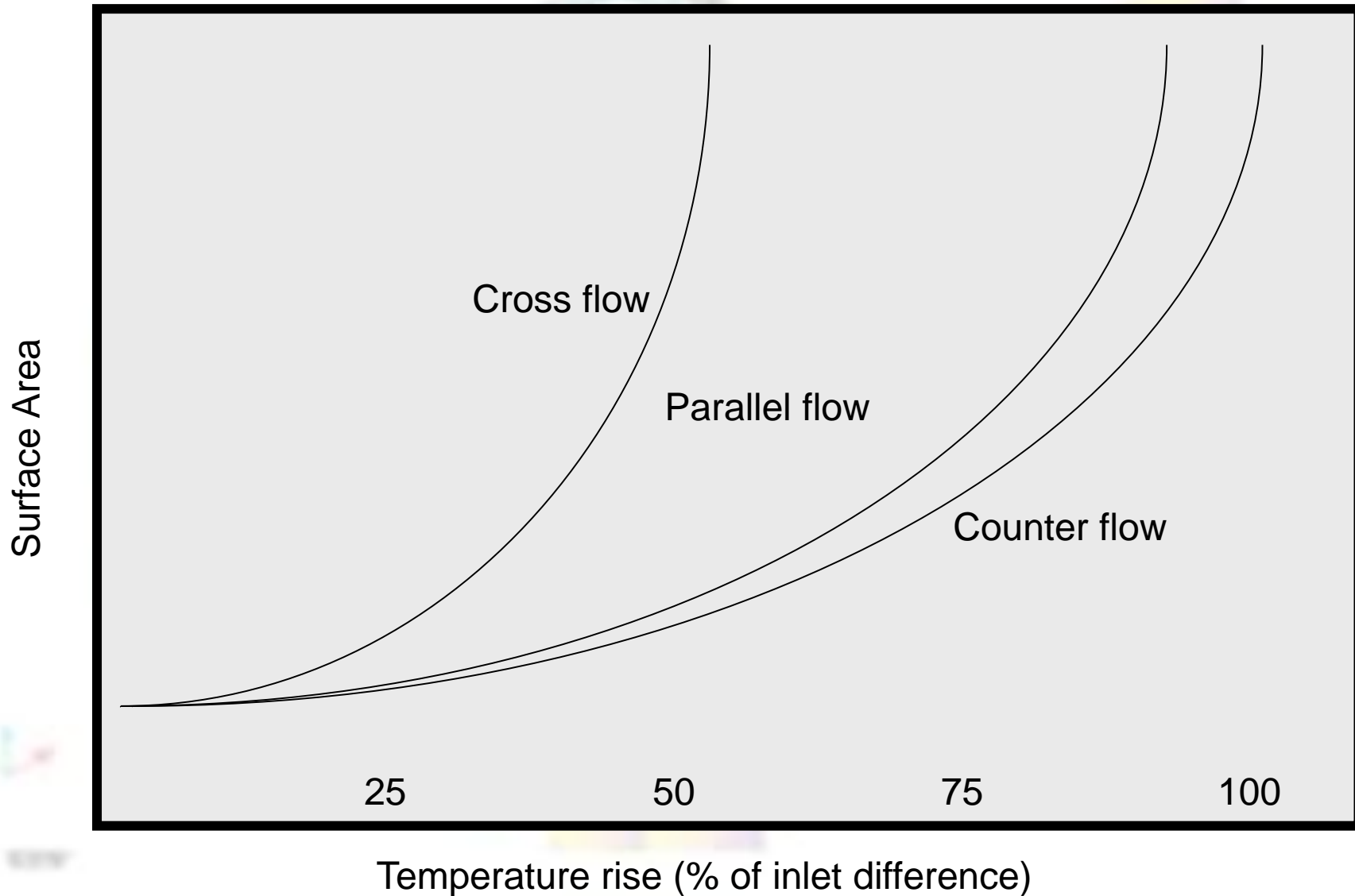


Parallel / Counter / Cross Flow

- Cross Flow – two fluids flow at right angles to each other



How Different Flow Patterns Compare



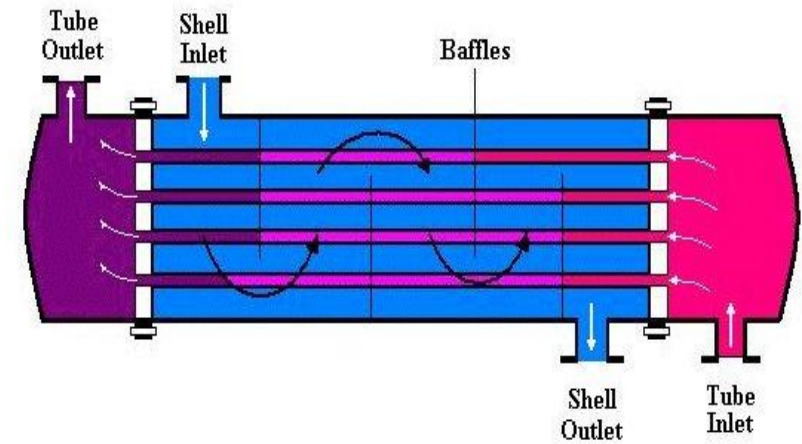
Double Pipe Heat Exchanger

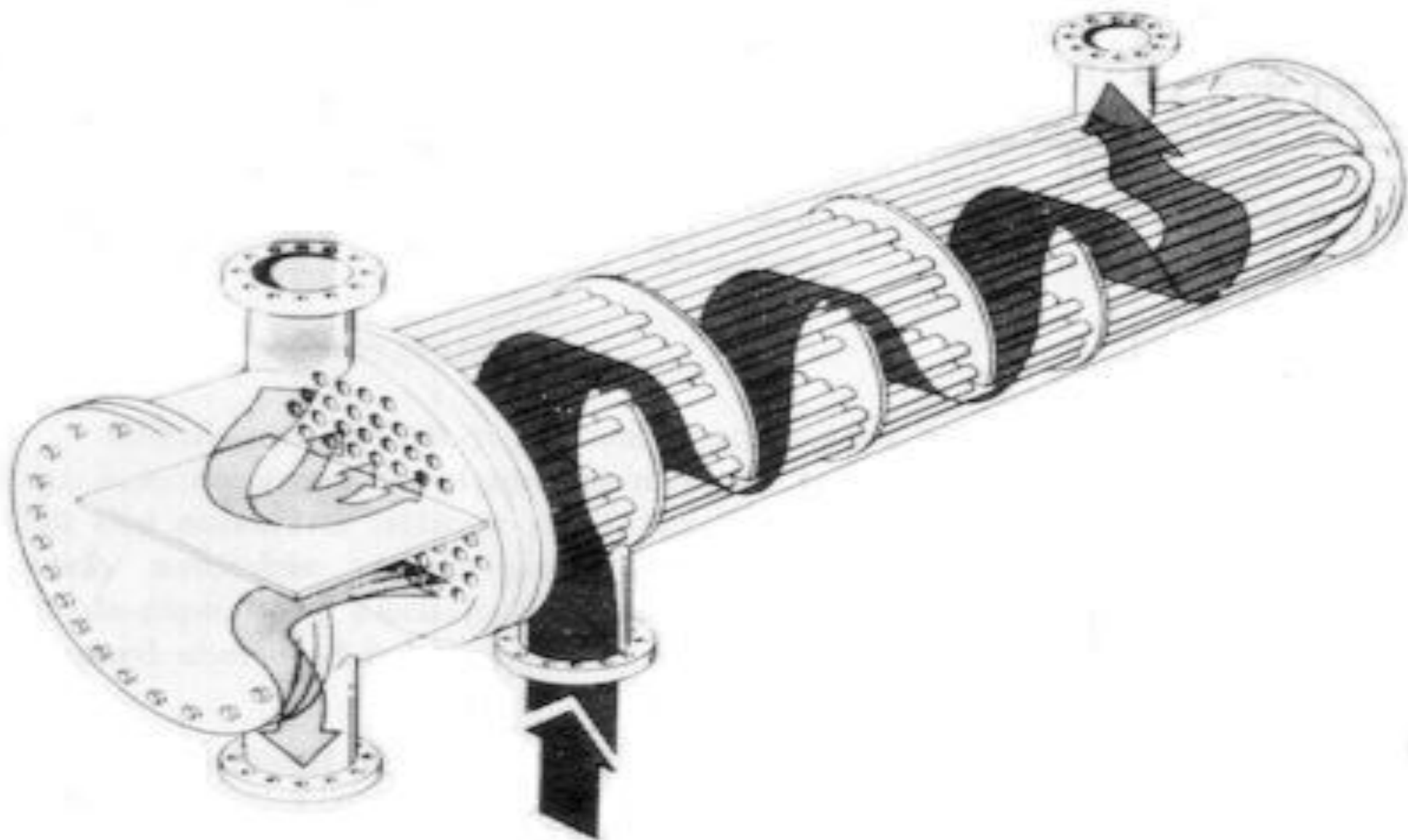
- One pipe located concentrically inside a second, larger one
- Have to be large to be effective



Shell & Tube Exchangers

- Most common HE (80% of market)
- One fluid in network of tubes, second surrounding in shell
- Baffles to complicate flow, improve efficiency (straight, helical)
- If phase change, should be in shell
- Tubes usually doubled or quadrupled back (input and output at same end)
- Tubes ribbed on inner and outer surfaces

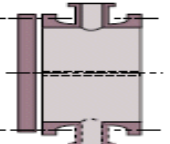


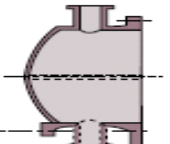
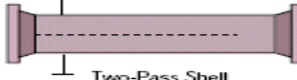

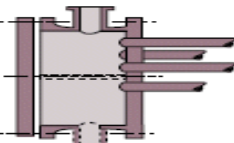
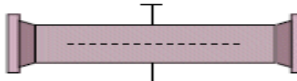

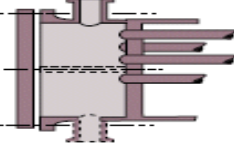


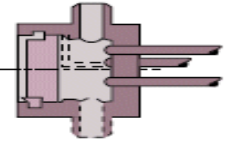
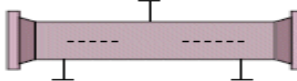


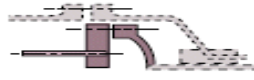
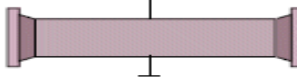

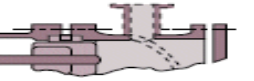




Shell & Tube Exchangers

- Specified by three letter code from TEMA
- Put high pressure (up to 100MPa) or corrosive fluids in tubes
- Put fluids prone to fouling in tubes (higher velocity = less build-up)
- Get lower pressure drop in tubes
- Vibration can be problem – avoid resonance
- Temperature stresses important – U shaped bundles good

3 Letter TEMA Codes

Stationary Head Types		Shell Types		Rear Head Types	
A	 Removable Channel and Cover	E	 One-Pass Shell	L	 Fixed Tube Sheet Like "A" Stationary Head
B	 Bonnet (Integral Cover)	F	 Two-Pass Shell with Longitudinal Baffle	M	 Fixed Tube Sheet Like "B" Stationary Head
C	 Integral With Tubesheet Removable Cover	G	 Split Flow	N	 Fixed Tube Sheet Like "C" Stationary Head
N	 Channel Integral With Tubesheet and Removable Cover	H	 Double Split Flow	P	 Outside Packed Floating Head
D	 Special High-Pressure Closures	J	 Divided Flow	S	 Floating Head with Backing Device
		K	 Kettle-Type Reboiler	T	 Pull-Through Floating Head
		X	 Cross Flow	U	 U-Tube Bundle
				W	 Externally Sealed Floating Tubesheet

Source:

CHEMICAL ENGINEERING PROGRESS • FEBRUARY 1998

Heat Exchanger Analysis

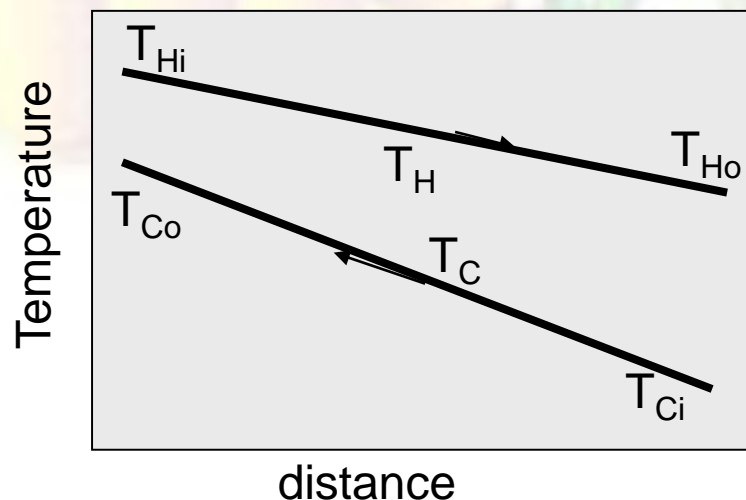
- Four temperatures
- Two flow rates
- Two heat capacities
- Conductivity of separating wall
- Area of separating wall
- Fouling
- Type of flow (cross/parallel/counter)

Conservation of Heat

- Heat lost by hot fluid = heat gained by cold fluid
- Heat change
 - $Q = mc\Delta T$ for change in temperature
 - $Q = mL_{v,f}$ for change of phase
- $mc = C$

Log Mean Temperature Difference

- For conduction – $Q/t = UA \Delta T$ where ΔT is the difference in temperatures
- But for a heat exchanger ΔT changes along the length of the device
- For example, for a counter flow the profile is shown below
- So what value of ΔT should we use in the equation?



$$\Delta T_{lm} = \frac{[(T_{H_o} - T_{C_i}) - (T_{H_i} - T_{C_o})]}{\log_e \frac{(T_{H_o} - T_{C_i})}{(T_{H_i} - T_{C_o})}} \quad \text{(counter flow)}$$

- is known as the log mean temperature difference.
- It reflects an average value of the temperature difference across the Heat Exchanger
- For crossflow or multipass heat exchangers we use the same expression but introduce a correction term:

$$\Delta T_{lm} = F \frac{[(T_{H_o} - T_{C_i}) - (T_{H_i} - T_{C_o})]}{\log_e \frac{(T_{H_o} - T_{C_i})}{(T_{H_i} - T_{C_o})}} \quad \text{(cross flow)}$$

- F usually found from graphs (use $F = 0.9$ when graphs not available)
- For parallel flow the equation is:

$$\Delta T_{lm} = \frac{[(T_{H_o} - T_{C_o}) - (T_{H_i} - T_{C_i})]}{\log_e \frac{(T_{H_o} - T_{C_o})}{(T_{H_i} - T_{C_i})}} \quad \text{(parallel flow)}$$

- In general get:

$$\Delta T_{lm} = F \frac{\Delta T_2 - \Delta T_1}{\log_e \frac{\Delta T_2}{\Delta T_1}}$$

- Where ΔT_2 is the temperature difference at one end of the pipe, ΔT_1 is the temperature difference at the other end and F is a correction factor read from tables

Problem on LMTD

- A parallel flow double pipe heat exchanger heats 2.52kg/s of water from 21.1°C to 54.4°C using hot water under pressure entering at 110°C and leaving at 70°C. The heat exchanger area is $A = 4.3\text{m}^2$ and the heat capacity of water is $4180\text{Jkg}^{-1}\text{°C}^{-1}$. Calculate the ΔT_{lm} in the exchanger and the overall heat transfer coefficient U .

[42.1°C, 1938Wm⁻²K⁻¹]

- A counter flow double pipe heat exchanger heats water from 15°C to 74.2°C using hot water entering at 91.3°C and leaving at 62.2°C. The heat exchanger area is $A = 8.51\text{m}^2$ and the overall heat transfer coefficient $U = 2500\text{Wm}^{-2}\text{K}^{-1}$. Calculate the ΔT_{lm} in the exchanger, q , and the flow rates in both pipes.

[29.3°C, 6.25x10⁵W, 2.52kg/s for the cold water, 5.14kg/s for the hot]

- A parallel flow double pipe heat exchanger heats water from 24°C to 44.9°C using hot water entering at 95°C. The heat exchanger area is $A = 4.4\text{m}^2$. The flow rates are 1.14kg/s for the cold water and 3.1kg/s for the hot water. Calculate the temperature of the hot water leaving the exchanger, the LMTD, and the overall heat transfer coefficient U .

[83.8°C, 53.4°C, 4238Wm⁻²K⁻¹]

Fouling



Fouling

- A €5bn problem (mostly in petrochem industry)
- Estimated to account for 3% of CO₂ emissions
- Problems
 - Reduced heat transfer
 - Reduced throughput
 - Increased pressure drop
 - Replacement costs
 - Cleaning costs and disposal of toxic wastes

Fouling - Causes



- Dirt, soot scale
- Coke from petroleum industry
- Corrosion products
- Algae and other biological growths (prevented by copper)
- Sedimentation – if fluid is a suspension. Self limiting
- Inverse solubility – as fluid cools down
- Chemical reactions

Fouling - Prevention

- Avoid low fluid velocities (less than 1m/s)
- Avoid high fluid velocities (greater than 4m/s for water). Causes erosion
- Avoid big temperature differences